## Time-resolved nonlinear coupling between orthogonal flexural modes of a pristine GaAs nanowire - Supplementary Information

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## I. EQUATIONS OF MOTION

To derive the equations of motion for the two modes we treat the nanowire as an isotropic inextensible singly clamped Euler-Bernoulli beam. Following the approach used by Crespo da Silva and Glynn <sup>1,2</sup> we obtain two nonlinear equations describing the flexural vibrations of the beam, for the displacement  $\tilde{u}$ :

$$\begin{split} m\ddot{\tilde{u}} &+ \tilde{\eta_1}\dot{\tilde{u}} + D_1\tilde{u}'''' + D_1\left(\tilde{u}'\left(\tilde{u}'\tilde{u}'' + \tilde{w}'\tilde{w}''\right)'\right)' \\ &+ \left((D_1 - D_2)\left[\tilde{w}''\int_L^s \tilde{u}''\tilde{w}''\,ds - \tilde{w}'''\int_0^s \tilde{u}''\tilde{w}'\,ds\right] \\ &- \frac{(D_1 - D_2)^2}{D_k}\left[\tilde{w}''\int_0^s \int_L^s \tilde{u}''\tilde{w}''\,ds\,ds\right]'\right)' \\ &+ \frac{1}{2}m\left[\tilde{u}'\int_L^s \frac{\partial^2}{\partial\tilde{t}^2}\int_0^s \left(\tilde{u}'^2 + \tilde{w}'^2\right)\,ds\,ds\right]' = \tilde{Q_1} - \left(\tilde{u}'\int_L^s \tilde{Q_2}\,ds\right)' \end{split}$$
(1)

and a symmetric one for the displacement  $\tilde{w}$ . Here the dots and primes stand for derivative in time  $\tilde{t}$  and in arc length s respectively, m is the mass per unit length (=  $\rho d_1 d_2$  with  $d_1$ and  $d_2$  dimensions of the cross section and  $\rho$  the density),  $\tilde{\eta}_{1,2}$  the damping coefficient,  $D_{1,2}$ the bending, and  $D_k$  the torsional stiffnesses of the beam, and  $\tilde{Q}_{1,2}$  the generalized forces along the two directions. We define  $\tilde{F}_1 = \tilde{Q}_1 - \left(\tilde{u}'\int_L^s \tilde{Q}_2 ds\right)'$ . Eq. 1 can be rewritten in a dimensionless form <sup>3</sup> substituting  $u = \tilde{u}/d_1, w = \tilde{w}/d_1, x =$ 

Eq. 1 can be rewritten in a dimensionless form <sup>3</sup> substituting  $u = \tilde{u}/d_1, w = \tilde{w}/d_1, x = s/L, \eta_1 = \tilde{\eta}_1 L^4/(D_1 \tau)$  and scaling time with  $\tau = L^2 \sqrt{m/D_1}$ .

Applying the Galerkin method for the first mode in the two directions  $u(x,t) = a(t)\xi(x)$ and  $w(x,t) = b(t)\xi(x)$  with  $\xi(x)$  the first flexural mode shape, equal in both directions, we obtain:

$$\ddot{a} + \omega_1^2 a + \eta_1 \dot{a} + \alpha \left(\frac{d_1}{L}\right)^2 a^3 + \left(\alpha + \left(1 - \frac{D_2}{D_1}\right)\beta_1 - \frac{D_1}{D_k}\left(1 - \frac{D_2}{D_1}\right)^2\beta_2\right) \left(\frac{d_1}{L}\right)^2 ab^2 + \gamma \left(\frac{d_1}{L}\right)^2 a(\dot{a}^2 + a\ddot{a} + \dot{b}^2 + b\ddot{b}) = \varepsilon F_1$$
(2)

where  $F_1$  is the scaled dimensionless version of  $\tilde{F}_1$ , and

$$\alpha = \int_{0}^{1} \xi(x) \left( \xi'(x) \left( \xi'(x) \xi''(x) \right)' \right)' dx = 40.41$$

$$\beta_{1} = \left[ \int_{0}^{1} \xi(x) \left( \xi''(x) \int_{0}^{x_{1}} \left( \xi''(x) \right)^{2} dx_{1} \right)' dx \right]$$
(3)

$$= \left[ \int_{0}^{1} \xi(x) \left( \xi''(x) \int_{1}^{x} \left( \xi''(x) \int_{0}^{x_{1}} \xi''(x) \xi'(x) dx_{1} \right)' dx \right] - \left[ \int_{0}^{1} \xi(x) \left( \xi'''(x) \int_{0}^{x_{1}} \xi''(x) \xi'(x) dx_{1} \right)' dx \right] = -20.11$$
(4)

$$\beta_2 = \int_0^1 \xi(x) \left( \left( \xi''(x) \int_0^x \int_1^{x_1} \left( \xi''(x) \right)^2 dx_2 dx_1 \right)' \right)' dx = 16.60$$
(5)

$$\gamma = \int_0^1 \xi(x) \left(\xi'(x) \int_0^x \int_1^{x_1} \left(\xi'(x_2)\right)^2 dx_2 dx_1\right)' dx = 4.60 \tag{6}$$

$$\varepsilon = \int_0^1 \xi(x) \, dx = 0.78 \tag{7}$$

From our COMSOL simulations, the ~ 0.5 % difference in the cross section of the nanowire is already enough to produce a frequency splitting of the two perpendicular modes similar to what is observed, such that  $D_1/D_2 \simeq 1.01$ . We also consider the beam to have high torsional rigidity compared to the flexural rigidity, such that  $D_k \gg D_{1,2}$ . Finally, we find that nonlinear damping is negligible by evaluating the critical frequency at which the bistability of the Duffing regime starts to occur. As a result, we obtain the final simplified equation of motion in one direction:

$$\ddot{a} + \omega_1^2 a + \eta_1 \dot{a} + \alpha \left(\frac{d_1}{L}\right)^2 a^3 + \alpha \left(\frac{d_1}{L}\right)^2 a b^2 = \varepsilon F_1 \tag{8}$$

Note that, from equation 1 the coupling coefficient with dimensions would be defined as:

$$\tilde{\alpha}_1 \equiv \frac{D_1}{mL} \int_0^L \xi(s) \Big( \xi'(s) \big( \xi'(s) \xi''(s) \big)' \Big)' \, ds \tag{9}$$

## II. MECHANICAL LOGIC

Due to the Duffing nonlinearity, when sweeping the driving amplitude at fixed frequency  $f_2$ , we observe a high jump between two levels in the response amplitude of mode 2, at a critical driving amplitude. These two levels in the response are used to encode logical 0 and 1 output states<sup>4,5</sup>. The two inputs correspond to two signal voltages which are summed and subsequently applied to the driving PZT. Logical 0 and 1 input states are defined by low

and high driving voltages, respectively (See Fig. 1a). As shown in Fig. 1b (upper panel), we obtain a high response when one or both inputs are high (01, 10, or 11) and a low response when both input signals are low (00). This is therefore a realization of a logical OR gate. This OR gate is converted into a NOR gate by taking as output the response amplitude of mode 1 at  $f_1$  (lower panel Fig. 1b). When mode 2 is at the low level (for input 00) there is almost no interaction between the two modes and we have the maximum response (logical 1) of mode 1 at the readout frequency. When instead the amplitude of mode 2 is high (for 01, 10, and 11), mode 1 shifts to a higher frequency and the logical output is 0.



FIG. 1: (a) Simulated amplitude response curve of a Duffing oscillator, displaying its bistable regime of motion. Dashed black lines highlight the driving amplitudes needed to created OR and NOR gates. (b) Top panel: response amplitude of mode 2 as a function of time, for the four combinations of two logical inputs, as indicated by the numbers on top. Bottom panel: response amplitude of mode 1 as a function of time, for the same logical inputs.

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