

**Feedback Cooling**

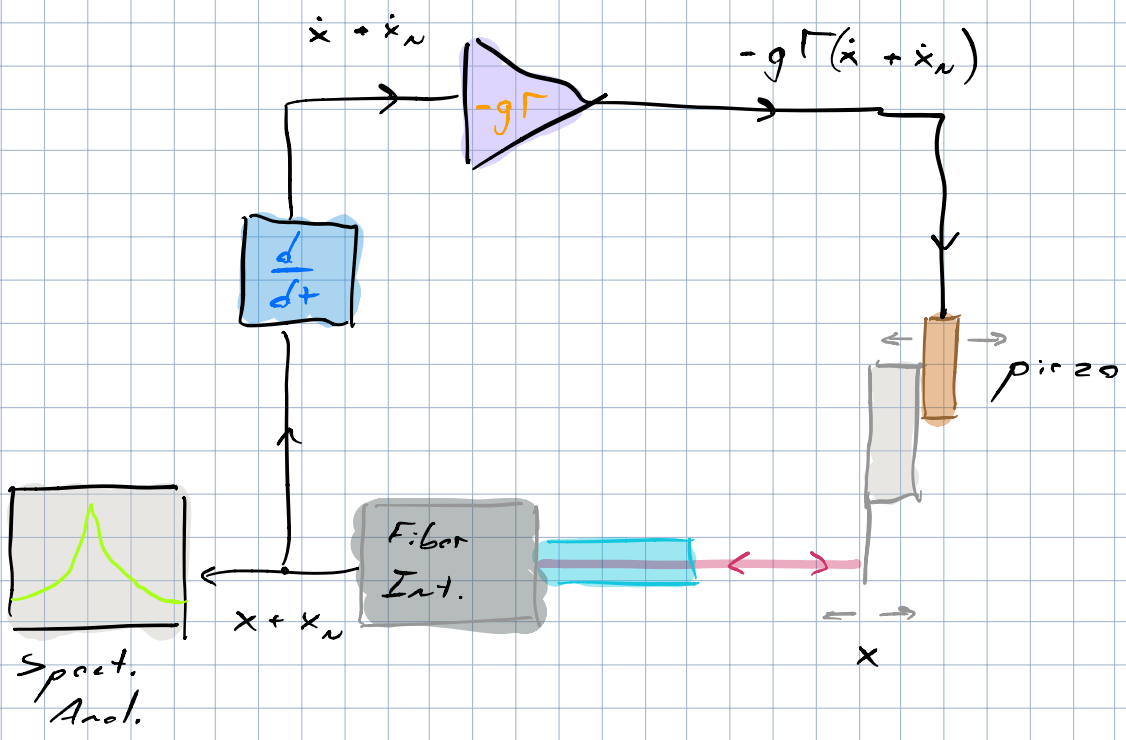
Take the equation of motion for the harmonic oscillator as our model for a mechanical mode:

$$m\ddot{x} + \Gamma\dot{x} + kx = F(t)$$

↖ thermal force

$$\Gamma = \frac{m\omega_0}{Q}, \quad k = m\omega_0^2$$

Detection setup and feedback setup:



Feedback gives a modified equation of motion:

$$m\ddot{x} + \Gamma\dot{x} + kx = F(t) - g\Gamma(\dot{x} + \dot{x}_N)$$

Let's look at one Fourier component:

$$-m\omega^2 \hat{x}(\omega) + i\omega\Gamma \hat{x}(\omega) + k\hat{x}(\omega) = \hat{f}(\omega) - ig\Gamma\omega [\hat{x}(\omega) + \hat{x}_n(\omega)]$$

$$\hat{x}(\omega) = \frac{\hat{f}(\omega) - ig\Gamma\omega \hat{x}_n(\omega)}{(k - m\omega^2) + i\omega\Gamma(1+g)}$$

Since  $\left| \frac{a+ib}{c+id} \right|^2 = \frac{a^2+b^2}{c^2+d^2}$  and  $S_x(\omega) = \lim_{T \rightarrow \infty} \frac{S(\omega)\hat{x}^*(\omega)}{T}$ :

$$\bar{S}_x(\omega) = \frac{\bar{S}_f(\omega) + g^2\Gamma^2\omega^2 \bar{S}_{x_n}(\omega)}{(k - m\omega^2)^2 + \omega^2\Gamma^2(1+g)^2}$$

$$\bar{S}_x(\omega) = \left[ \frac{1}{(k - m\omega^2)^2 + \omega^2\Gamma^2(1+g)^2} \right] \bar{S}_f(\omega)$$

← PSD of  
mode's displacement

$$+ \left[ \frac{g^2\Gamma^2\omega^2}{(k - m\omega^2)^2 + \omega^2\Gamma^2(1+g)^2} \right] \bar{S}_{x_n}(\omega)$$

What do we measure?

$$\hat{x}_{\text{meos}}(\omega) = \hat{x}(\omega) + \hat{x}_n(\omega)$$

$$\hat{x}_{\text{meos}}(\omega) = \frac{\hat{f}(\omega) - i g \Gamma \omega \hat{x}_n(\omega)}{(k - m\omega^2) + i \omega \Gamma (1 + g)} + \hat{x}_n(\omega)$$

$$\hat{x}_{\text{meos}}(\omega) = \frac{\hat{f}(\omega) + [(k - m\omega^2) + i \omega \Gamma] \hat{x}_n(\omega)}{(k - m\omega^2) + i \omega \Gamma (1 + g)}$$

Assume the thermal force noise  $\hat{f}(\omega)$  and the measured displacement noise  $\hat{x}_n(\omega)$  are uncorrelated. Therefore, the noises add in quadrature:

$$\bar{S}_{\text{meos}}(\omega) = \frac{\bar{S}_f(\omega) + [(k - m\omega^2)^2 + \omega^2 \Gamma^2] \bar{S}_{x_n}(\omega)}{(k - m\omega^2)^2 + \omega^2 \Gamma^2 (1 + g)^2}$$

$$\bar{S}_{\text{meos}}(\omega) = \left[ \frac{1}{(k - m\omega^2)^2 + \omega^2 \Gamma^2 (1 + g)^2} \right] \bar{S}_f(\omega) + \left[ \frac{(k - m\omega^2)^2 + \omega^2 \Gamma^2}{(k - m\omega^2)^2 + \omega^2 \Gamma^2 (1 + g)^2} \right] \bar{S}_{x_n}(\omega)$$

← PSD of measured signal

↳ Recall :  $\bar{S}_f(\omega) = 4k_B T \Gamma$

$\bar{S}_{x_n}(\omega) = \text{constant}$  ← typically from slot noise

By damping with gain  $g$ , we reduce the fluctuations in the mode of interest. This means that we cool the mode:

Recall :

$$\langle x^2 \rangle = \frac{1}{2\pi} \int_0^{\infty} \bar{S}_x(\omega) d\omega$$

Equipartition says :

$$\frac{1}{2} k_B T_{\text{mode}} = \frac{1}{2} k \langle x^2 \rangle$$

$$T_{\text{mode}} = \frac{k}{k_B} \langle x^2 \rangle$$

$$\therefore T_{\text{mode}} = \frac{k}{2\pi k_B} \int_0^{\infty} \bar{S}_x(\omega) d\omega$$

After integration ...

$$T_{\text{mode}} = \frac{T}{1+g} + \frac{k\Gamma}{4k_B m} \left( \frac{g^2}{1+g} \right) \bar{S}_{xN}$$

Minimize with respect to  $g$ :

$$T_{\text{mode, min}} = \sqrt{\frac{k\Gamma T}{k_B m} \bar{S}_{xN}} = \omega_0 \sqrt{\frac{\Gamma T}{k_B} \bar{S}_{xN}}$$

This is equivalent to a minimum phonon number:

$$N_{\text{mode, min}} = \frac{k_B T_{\text{mode, min}}}{\hbar \omega_0} = \frac{1}{\hbar} \sqrt{\Gamma k_B T \bar{S}_{xN}}$$

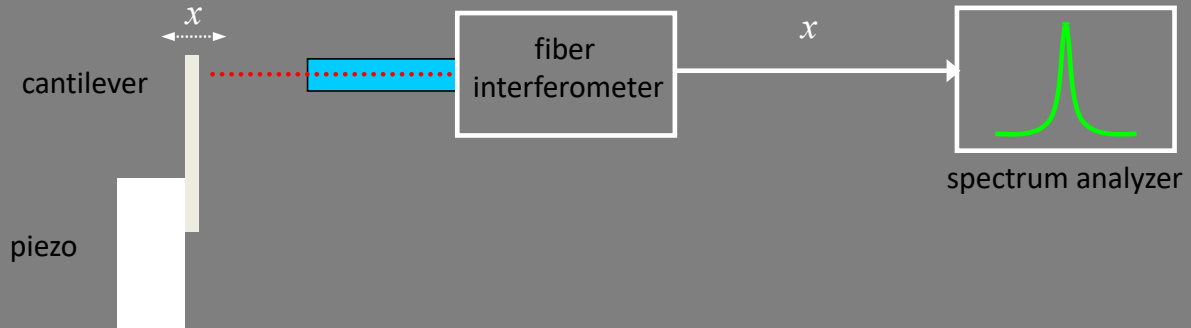
$$N_{\text{mode, min}} = \frac{1}{2\hbar} \sqrt{\bar{S}_F \bar{S}_{xN}} = \frac{1}{\hbar} \sqrt{S_F S_{xN}}$$

Optimum is achieved, as we will discuss, when  $\bar{S}_F$  is dominated by detector back-action and  $\bar{S}_{xN}$  is quantum limited. More on this to come...

Want low  $T$ ,  $\Gamma$ , and  $\bar{S}_{xN}$ . Cold fridge, good resonators, and sensitive measurement.

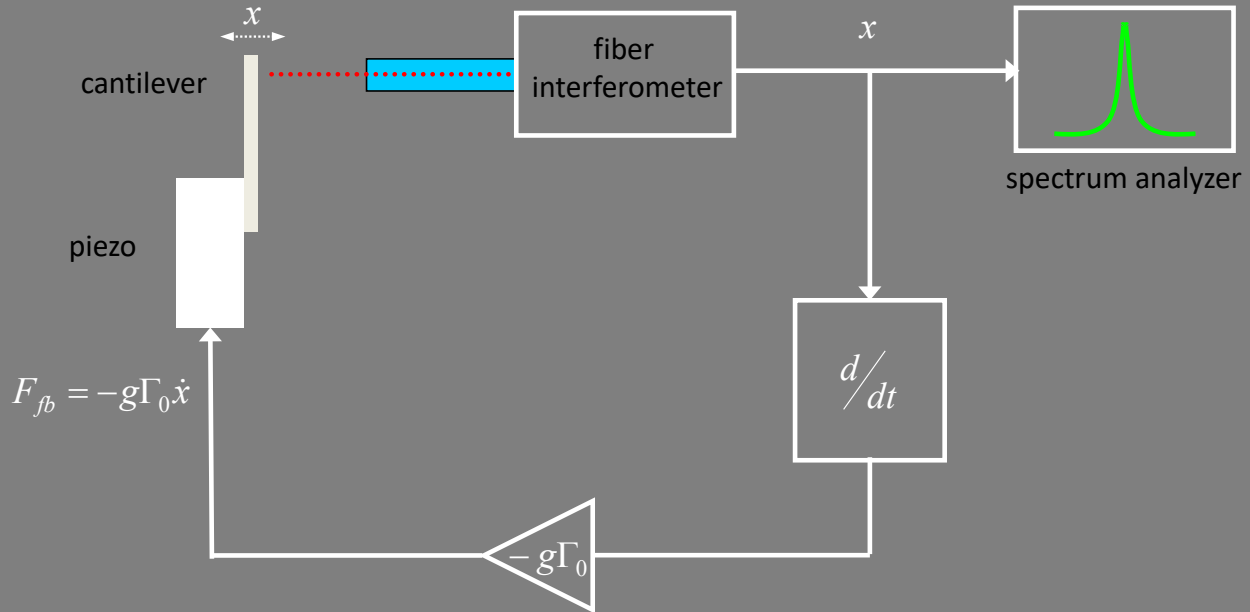
# Usual Cantilever Motion Detection

$$m\ddot{x} + \Gamma_0\dot{x} + kx = F_{th}$$

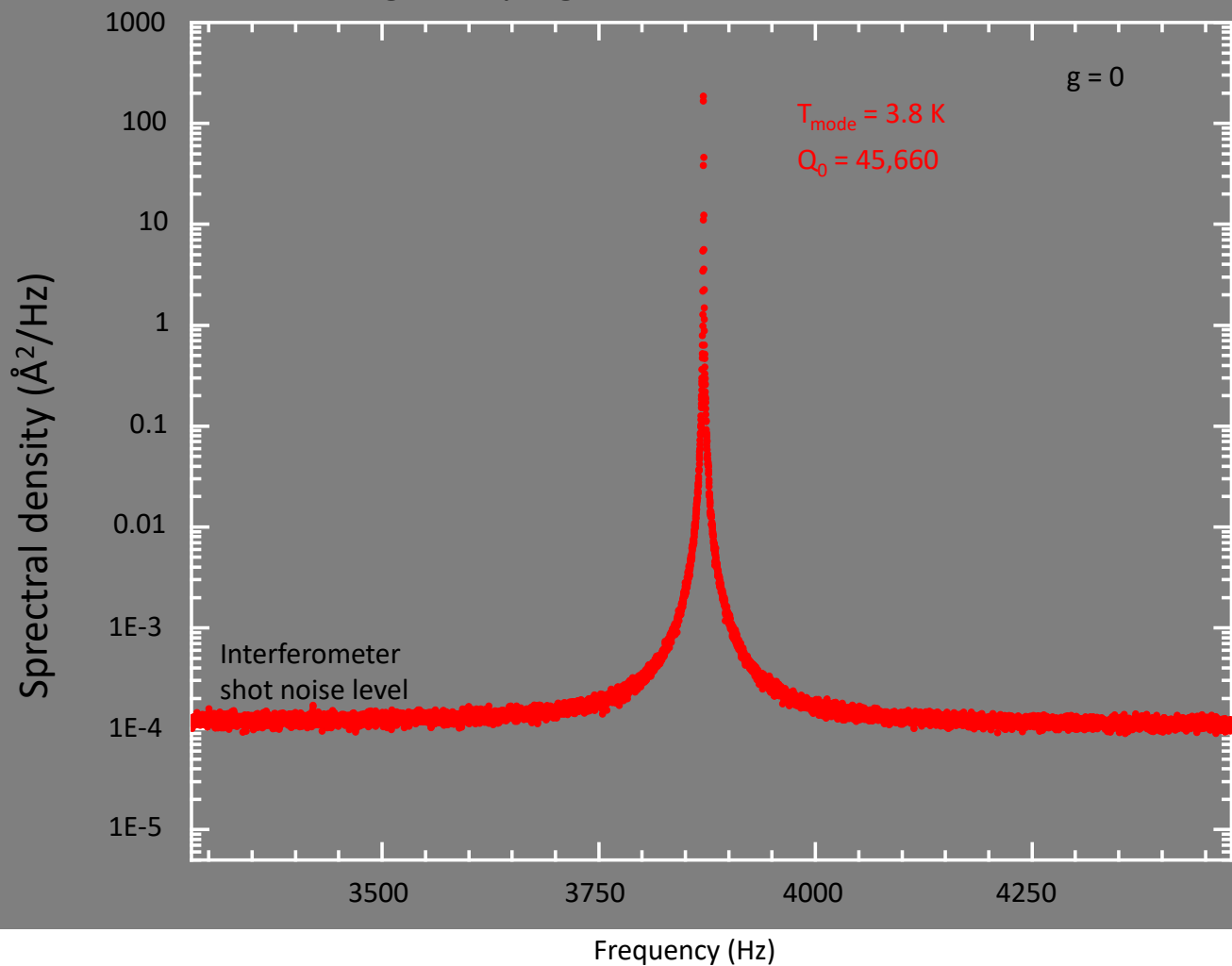


# Simple Electronic Damping

$$m\ddot{x} + \Gamma_0\dot{x} + kx = F_{th} - g\Gamma_0\dot{x}$$

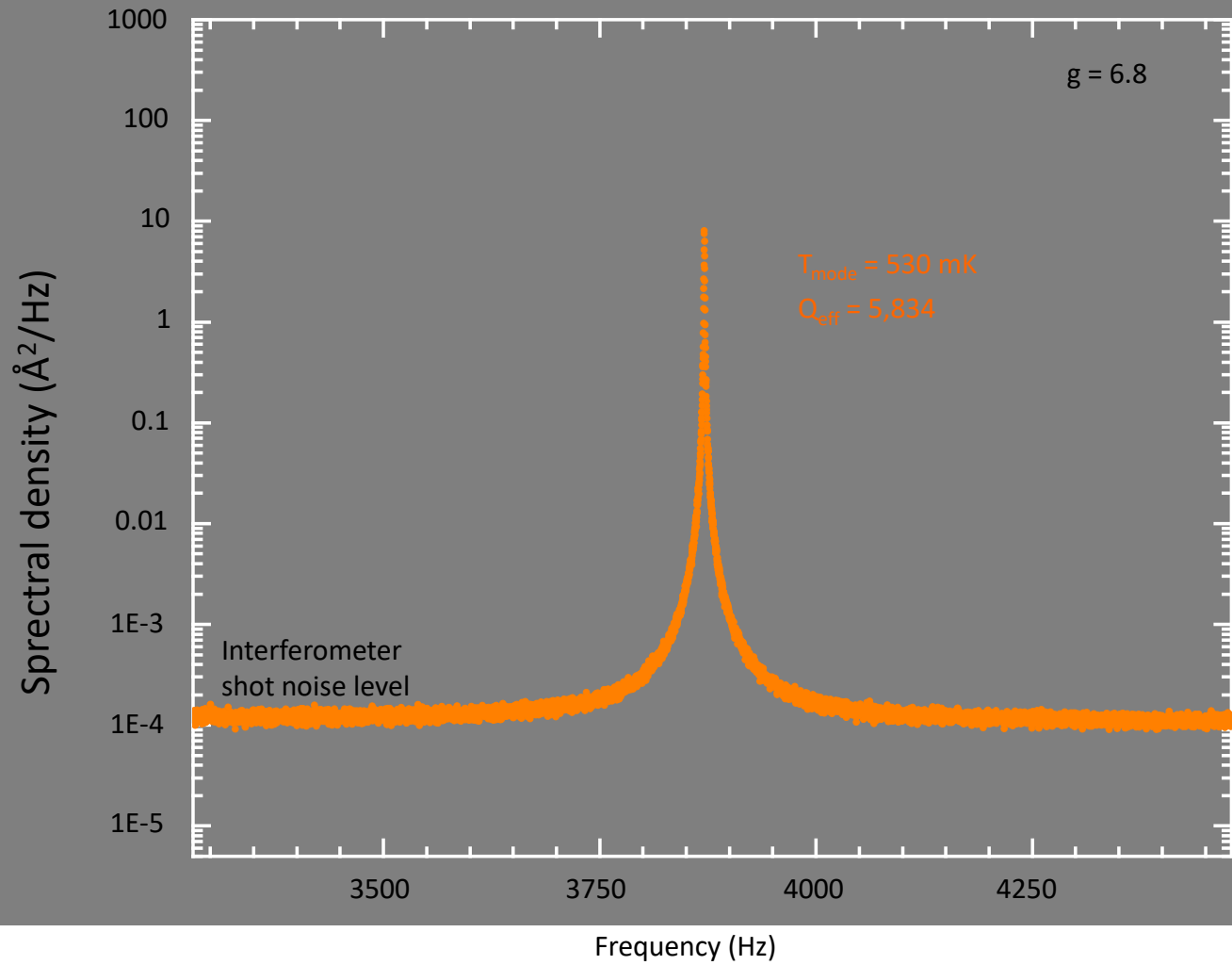


# Cooling (damping) of a cantilever - $T = 4.2\text{K}$

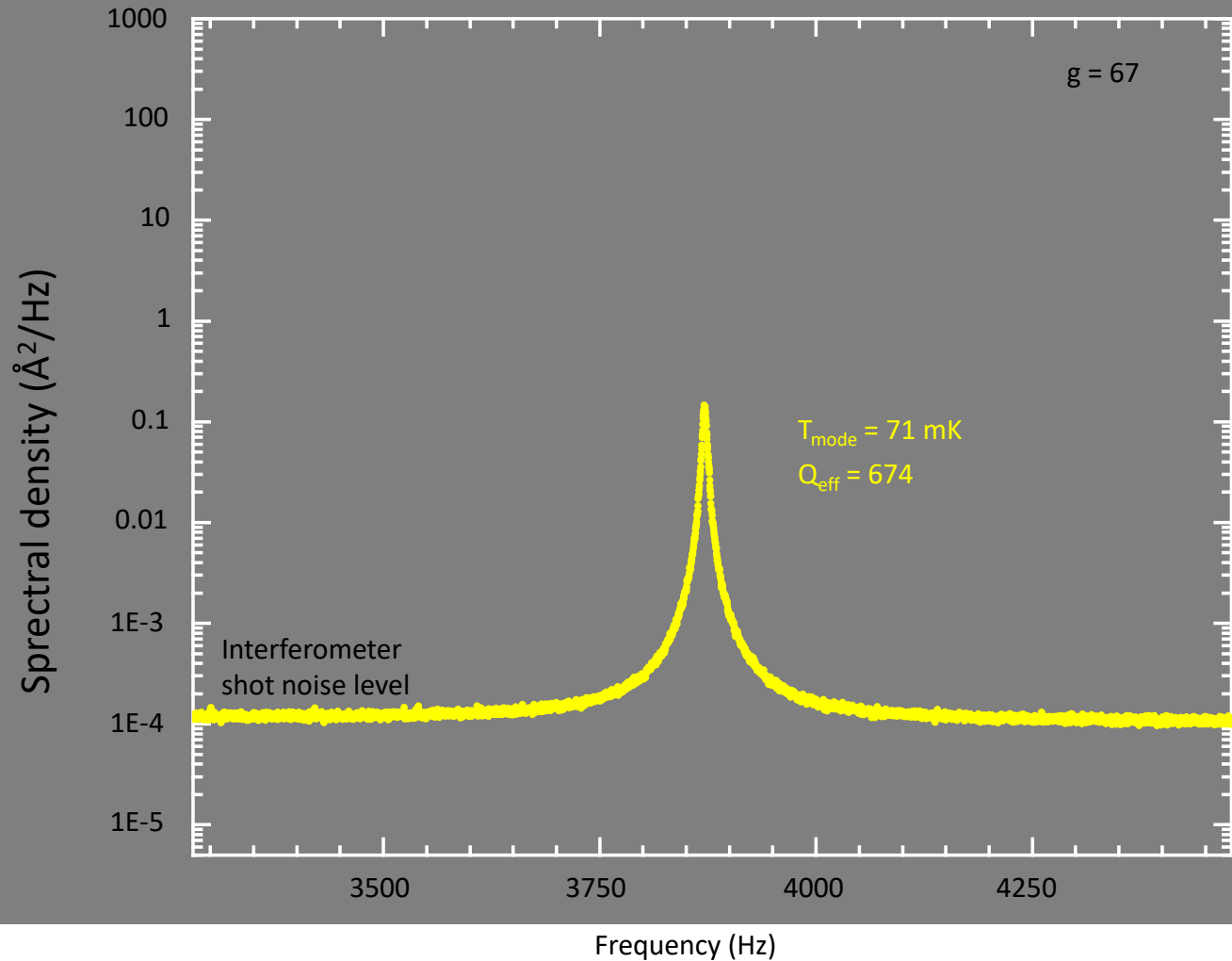




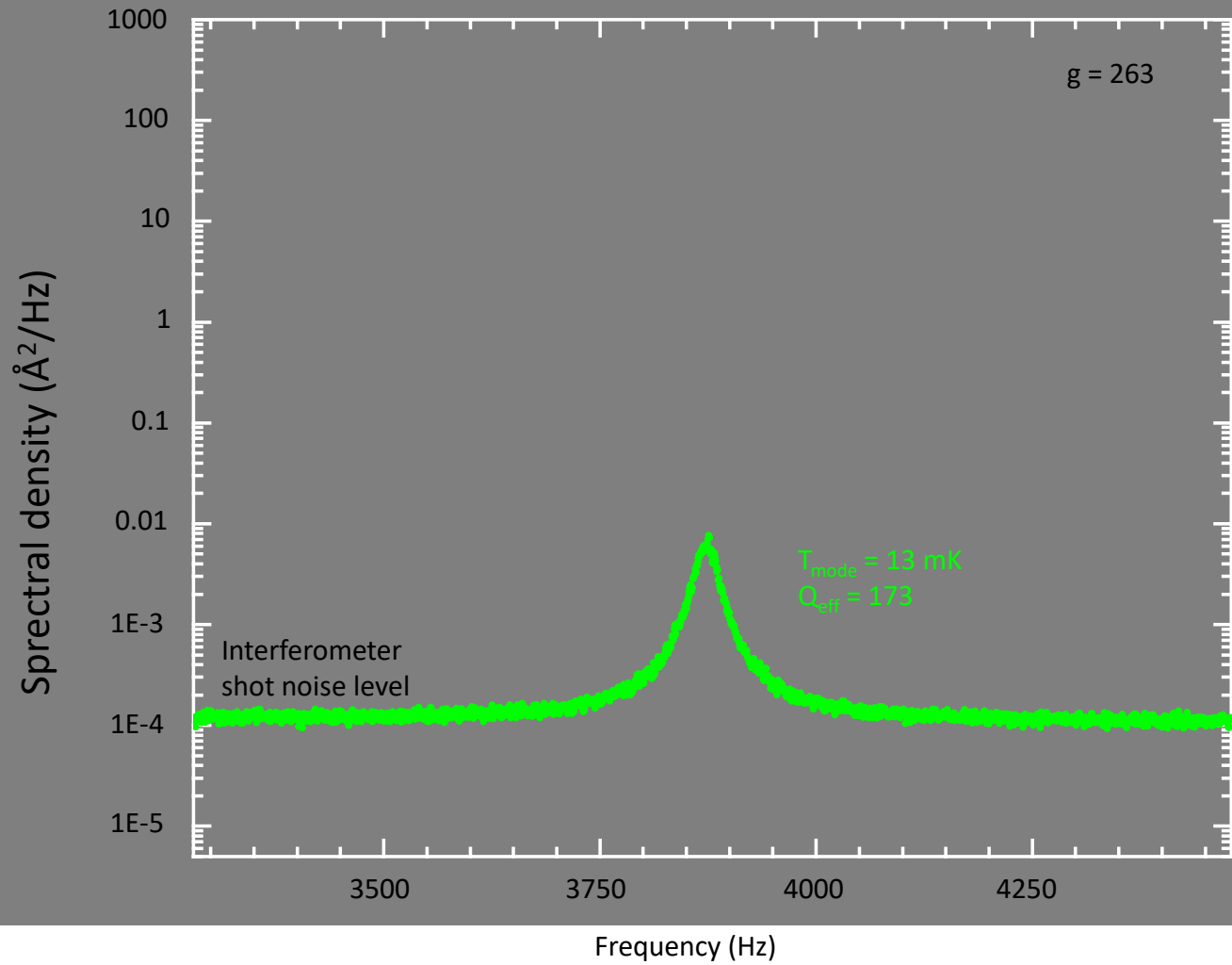
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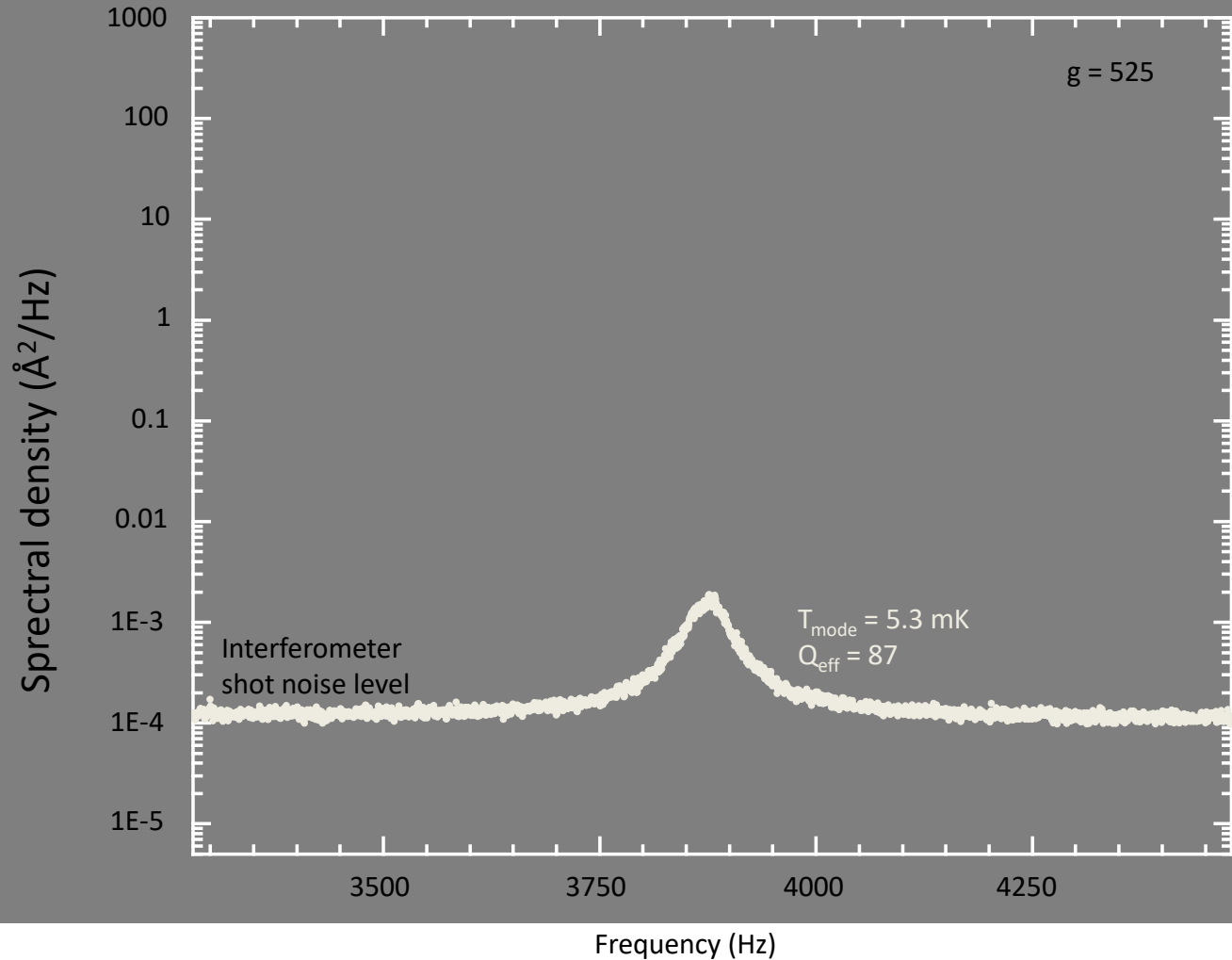
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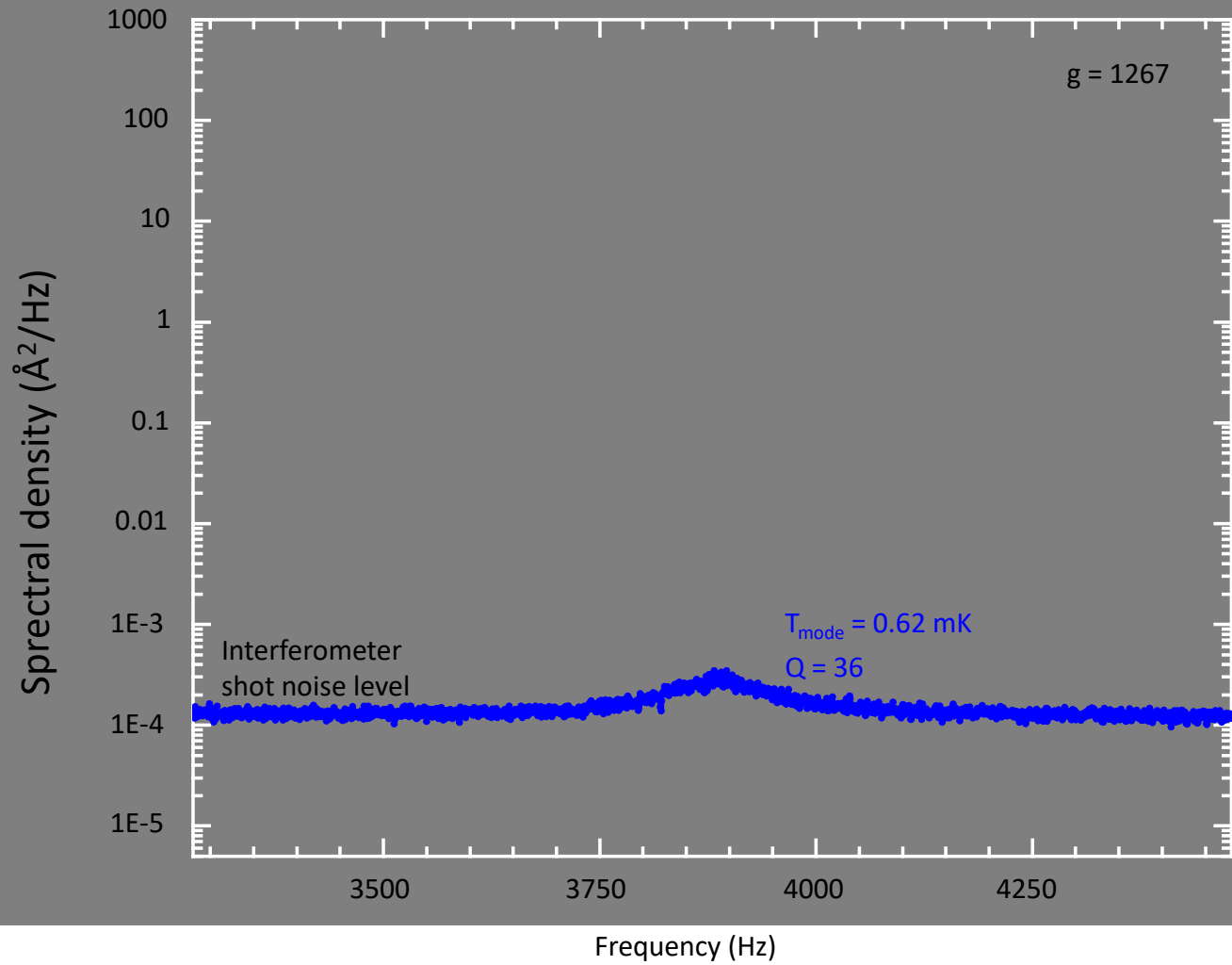
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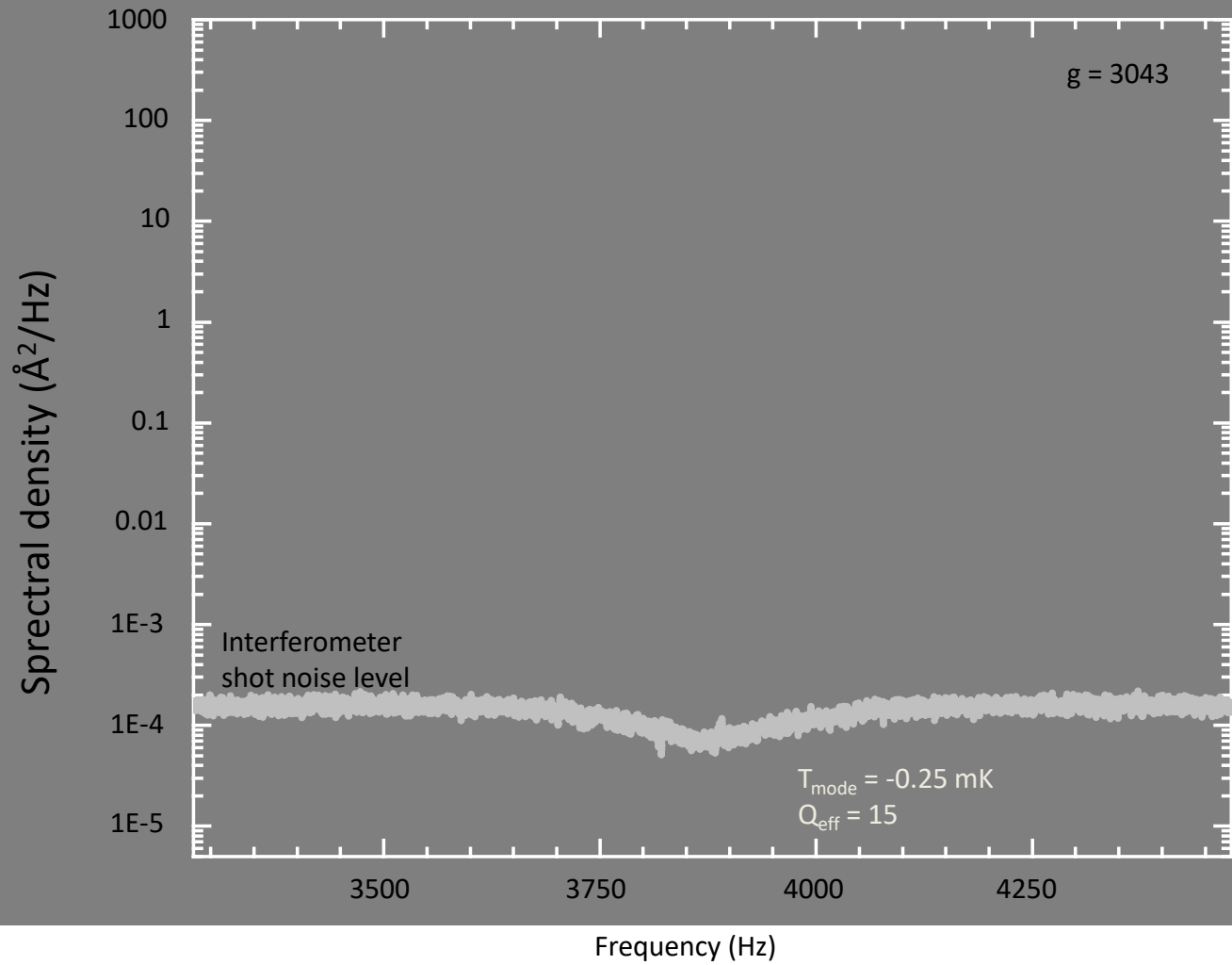
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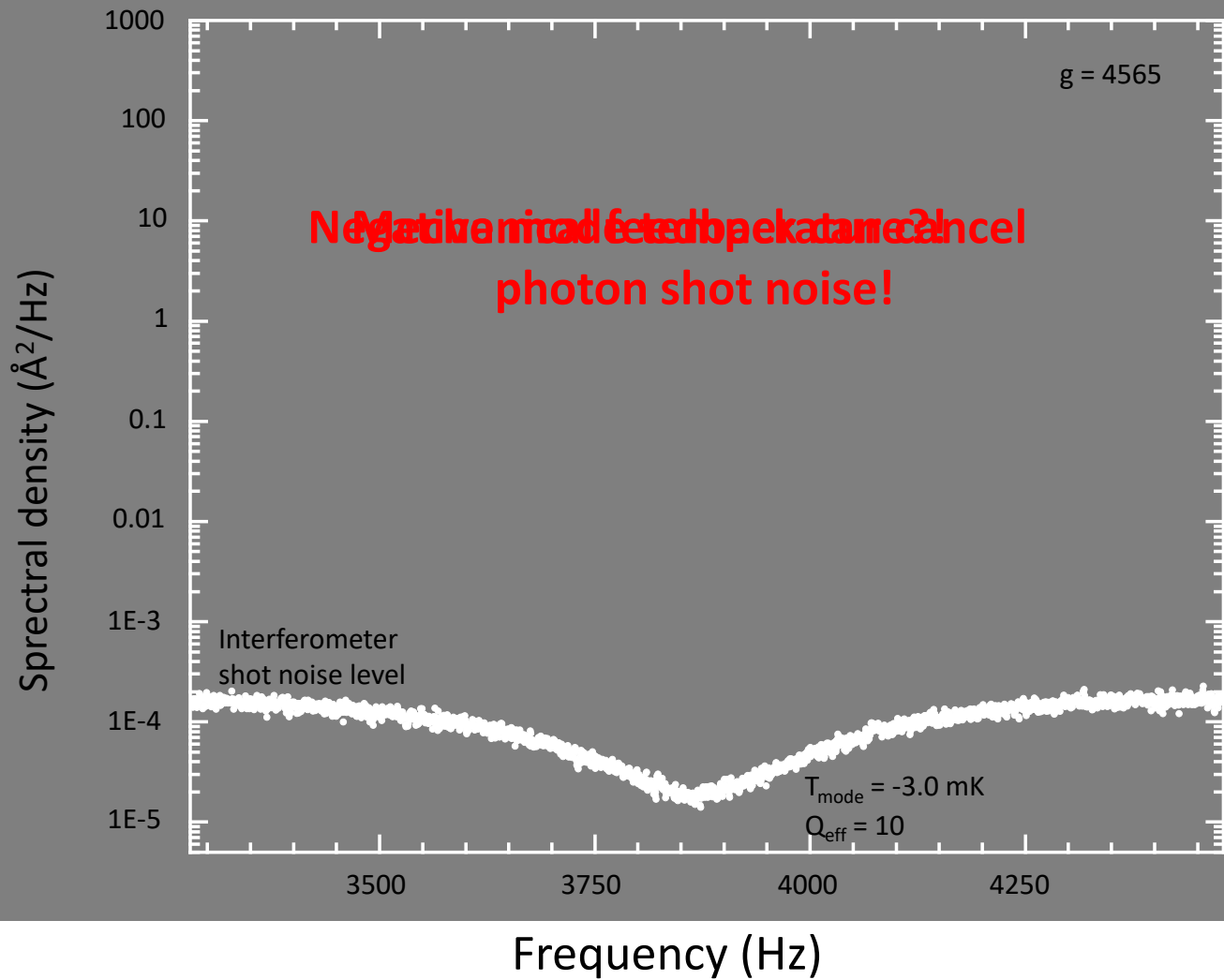
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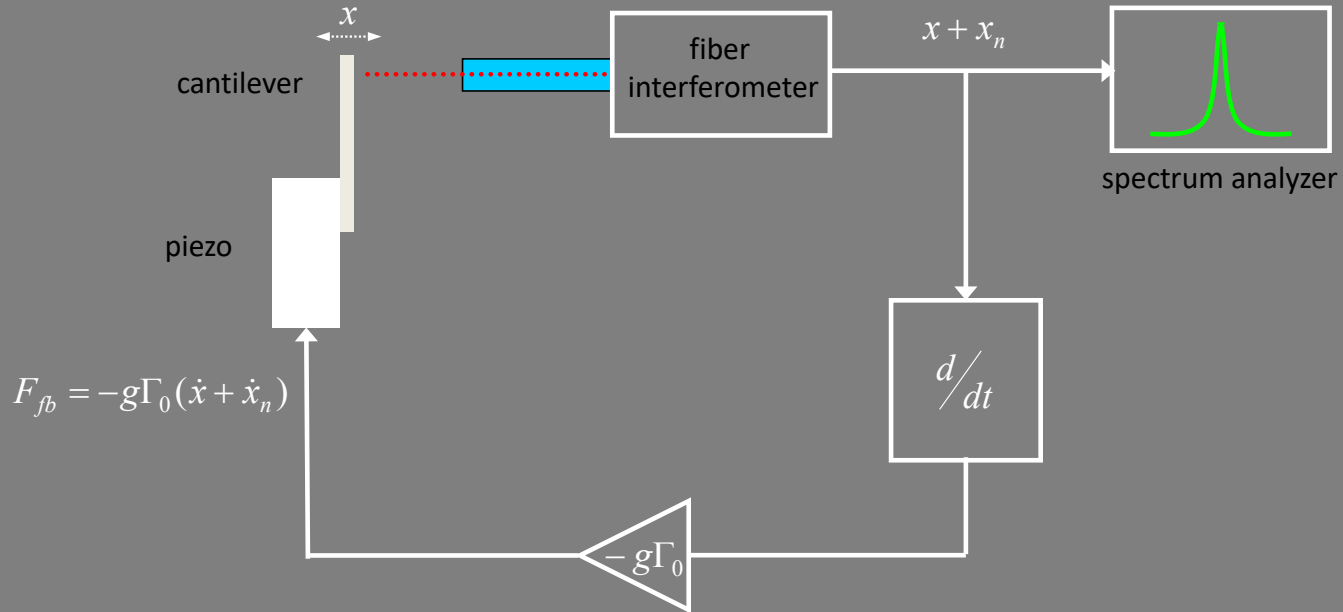


## Experimental setup

$$m\ddot{x} + \Gamma_0\dot{x} + kx = F_{th} - g\Gamma_0(\dot{x} - \dot{x}_n)$$

damping

measurement noise





## Cantilever Noise Temperature with Feedback

$$m\ddot{x} + \Gamma_0\dot{x} + kx = F_{th} - g\Gamma_0(\dot{x} + \dot{x}_n)$$

Effective Q with feedback:

$$Q_{eff} = \frac{Q_0}{1+g} = \frac{k}{\omega_c(1+g)\Gamma_0}$$

Measured spectral density:

$$S_{x+x_n}(\omega) = \left[ \frac{\omega_c^4/k^2}{(\omega^2 - \omega_c^2)^2 + \omega_c^2\omega^2/Q_{eff}^2} \right] S_{F_{th}} + \left[ \frac{(\omega^2 - \omega_c^2)^2 + \omega_c^2\omega^2/Q_0^2}{(\omega^2 - \omega_c^2)^2 + \omega_c^2\omega^2/Q_{eff}^2} \right] S_{x_n}$$

Actual cantilever spectral density:

$$S_x(\omega) = \frac{\omega_c^4/k^2}{(\omega^2 - \omega_c^2)^2 + \omega_c^2\omega^2/Q_{eff}^2} \left[ S_{F_{th}} + \frac{g^2k^2\omega^2}{\omega_c^2Q_0^2} S_{x_n} \right]$$

Cantilever mode temperature:

$$T_{mode} = \frac{k\langle x^2 \rangle}{k_B}$$

$$T_{mode} = \frac{T}{1+g} + \frac{1}{4} \frac{k}{k_B} \omega_c \left( \frac{g^2}{1+g} \right) \frac{1}{Q_0} S_{x_n}$$

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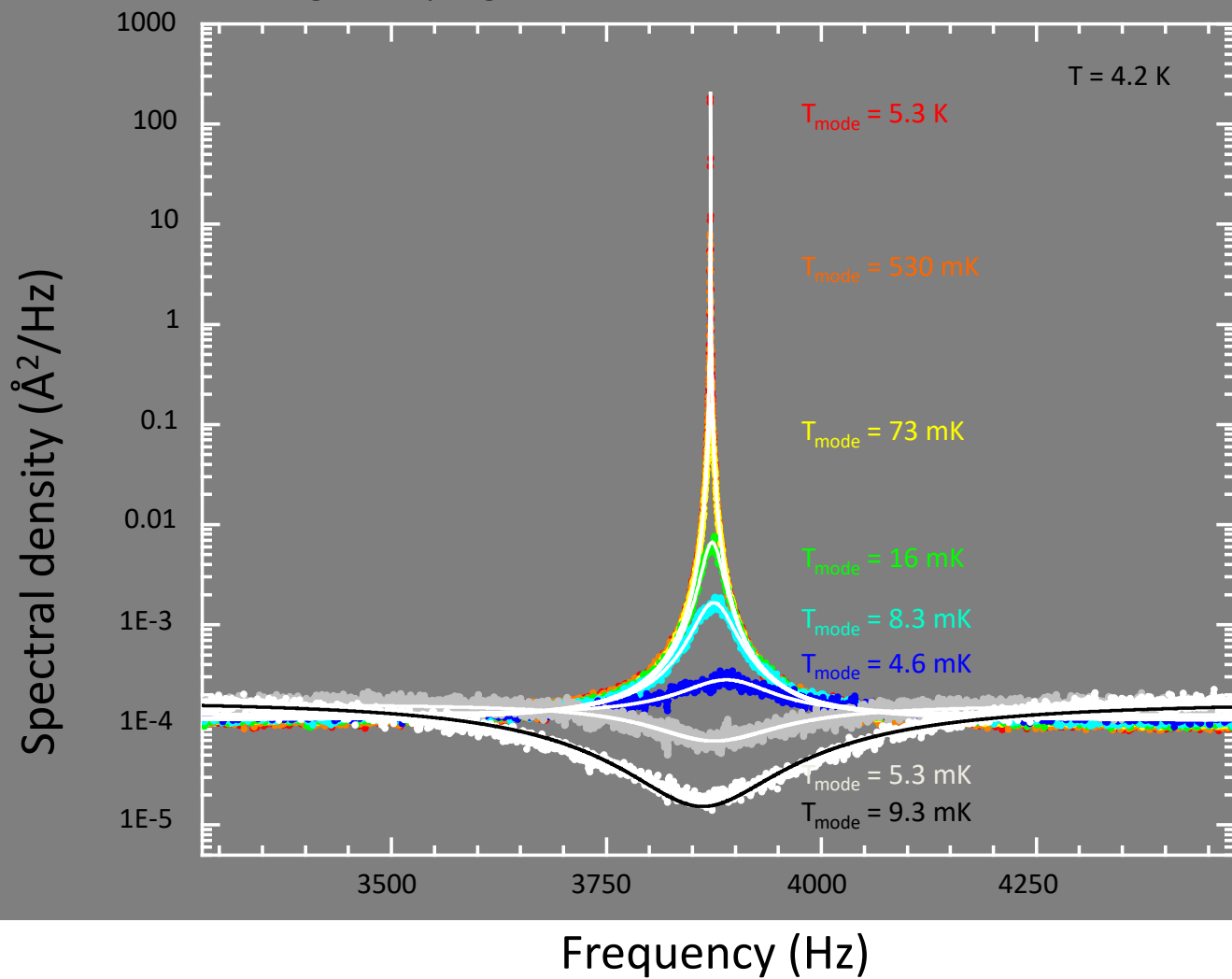
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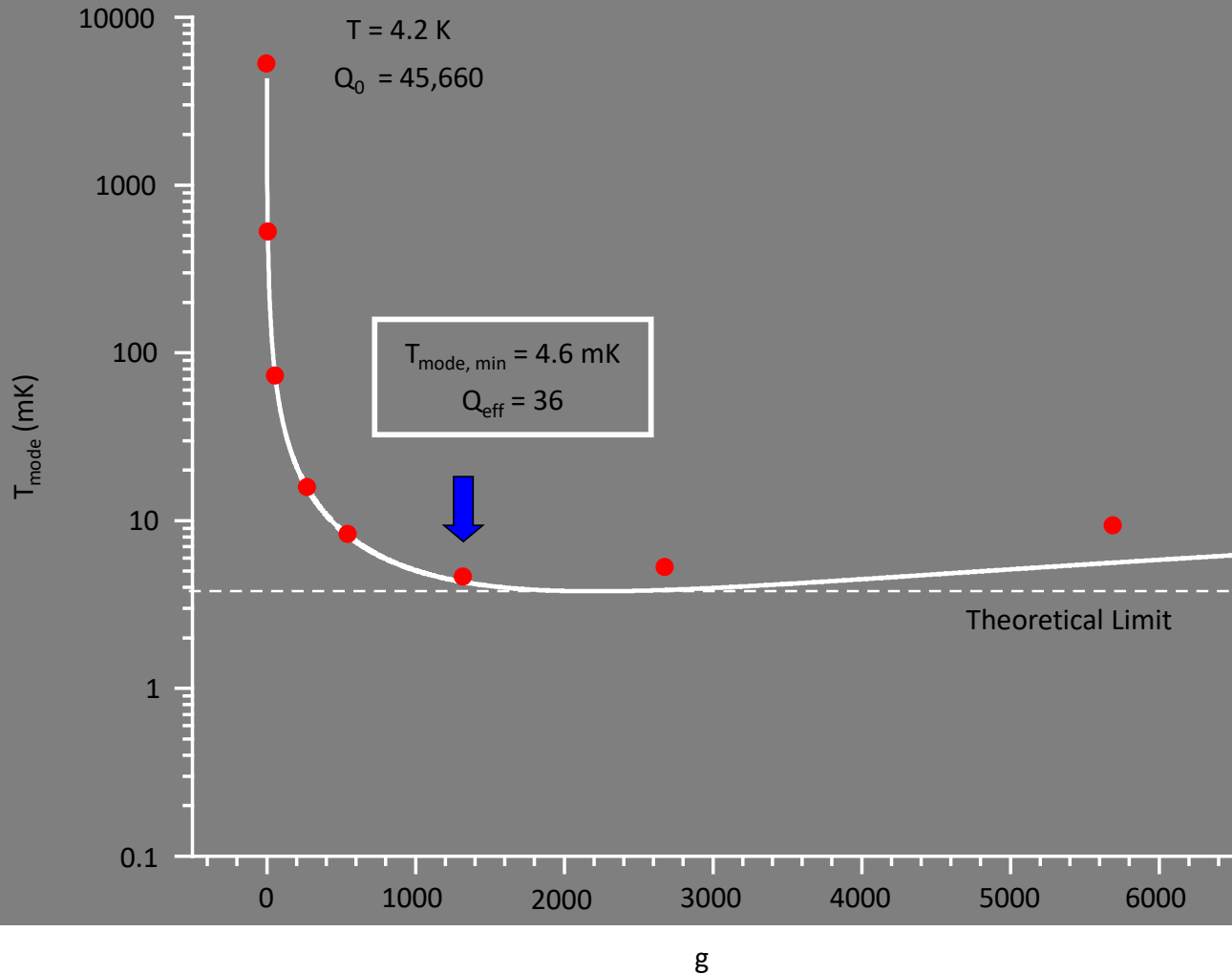
For optimum  
feedback gain

$$T_{mode,min} = \sqrt{\frac{k\omega_c T}{k_B Q_0}} S_{x_n}$$

# Cooling (damping) of a cantilever - $T = 4.2\text{K} \rightarrow 4.6\text{mK}$



# Cooling (damping) of a cantilever – model and experiment



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